



GEOHERMICA Initiative & CETPartnership TRI4  
Workshop in Dublin 10/10/2023

# Low-medium Temperature Geothermal for Campuses – University of Calgary, Canada

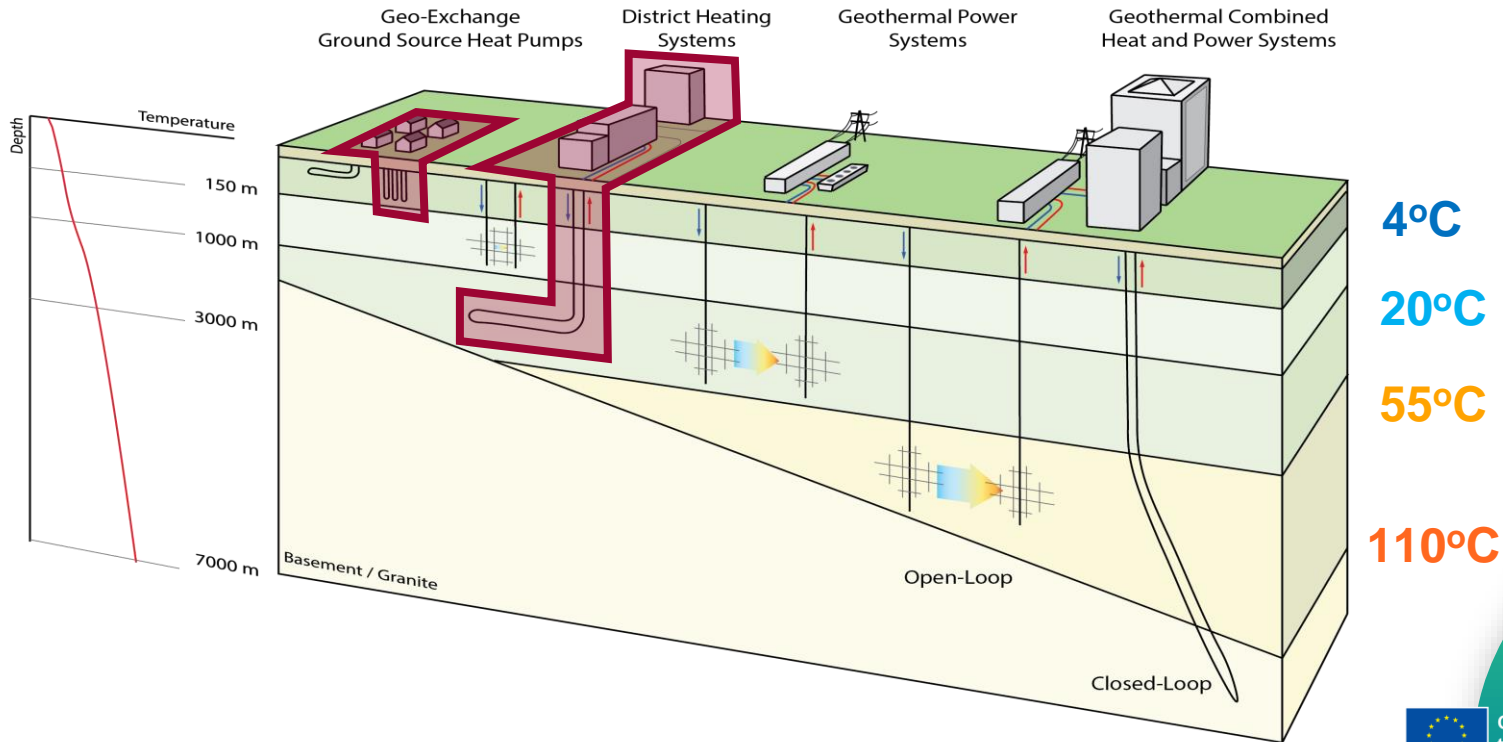
Aggrey Mwesigye, Ph.D., P.Eng



# Agenda

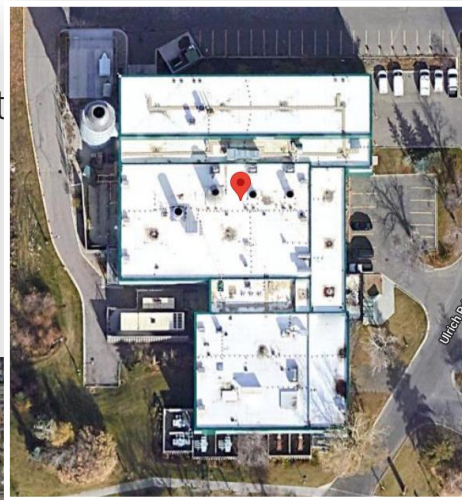
- Introduction
- University of Calgary District Heating System
- Proposed Geothermal System
- System Design
- Performance Results

# Introduction

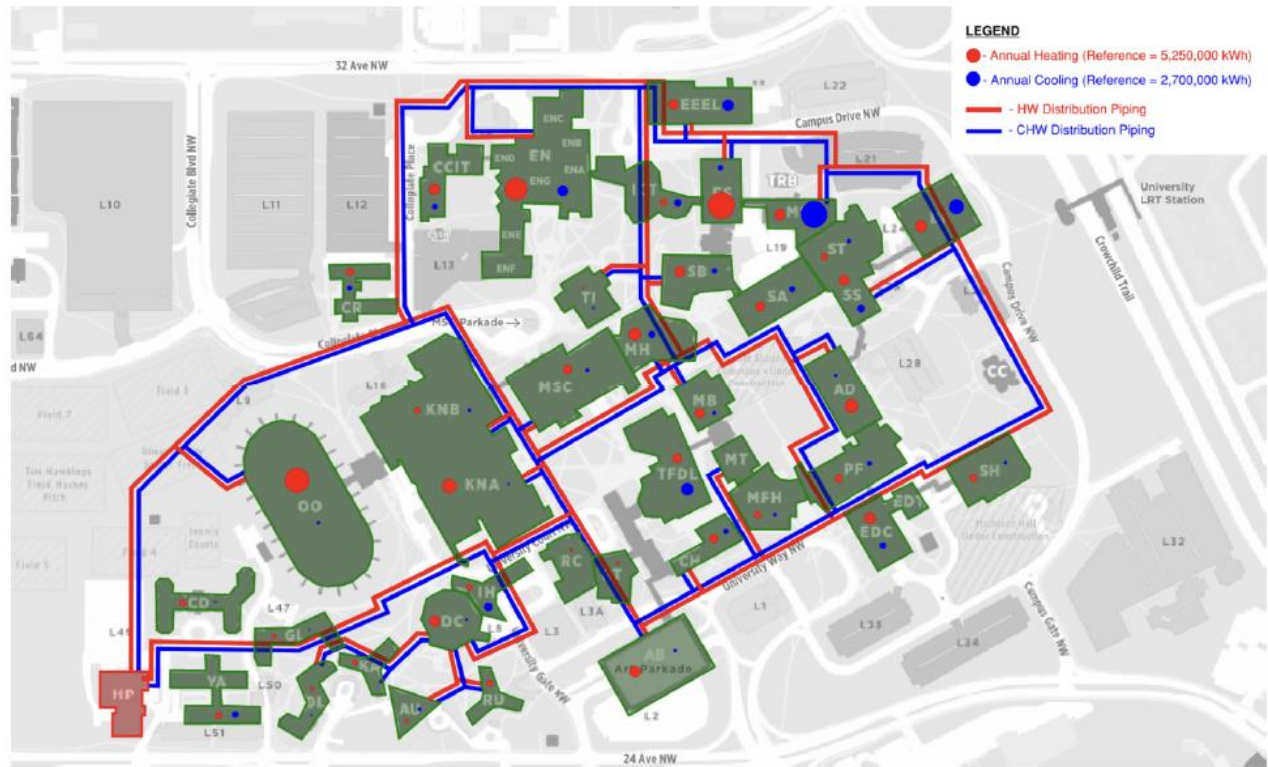


# University of Calgary System

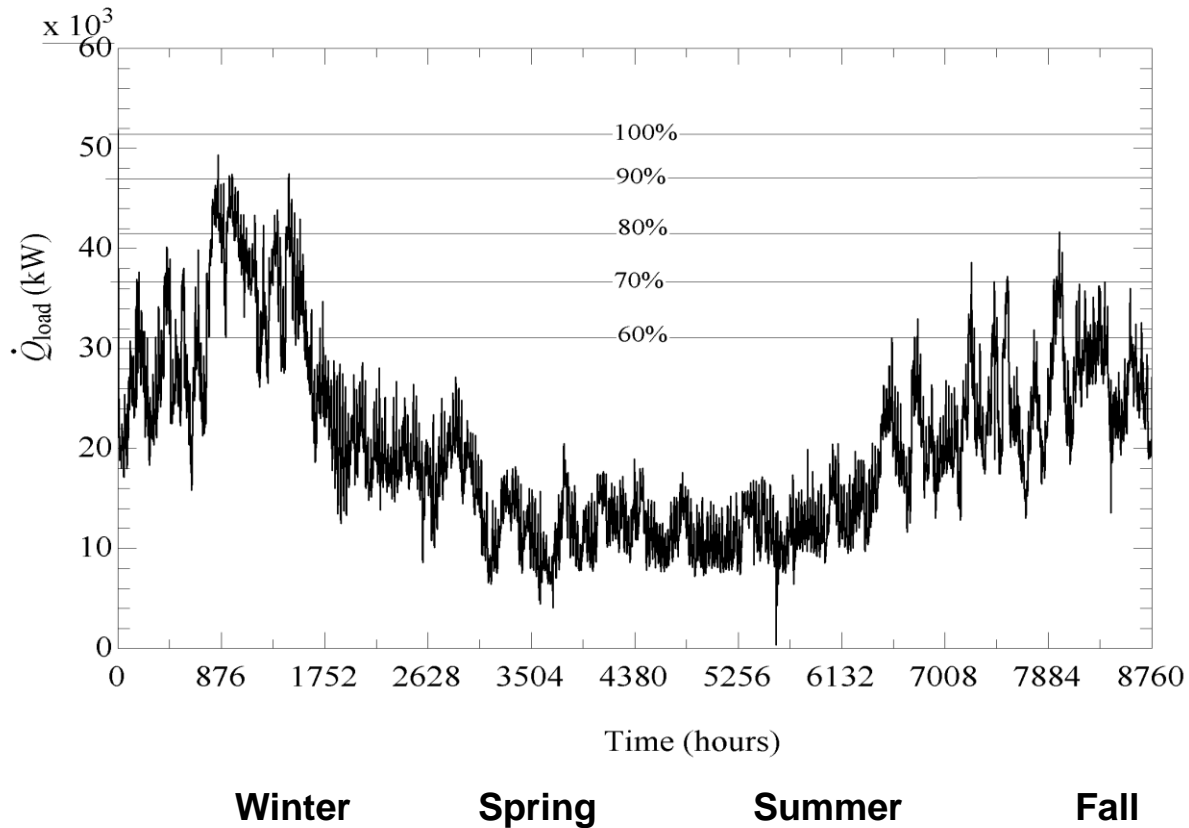
- Cogeneration system for combined heat and power (CHP)
  - 14 MW natural gas turbine – 70% of the electricity
  - Heat recovery system + supplementary boilers
  - District heating system
- Energy demand (annual)
  - Power: 85 GWhe/year
  - Heat: 147 GWht/year
- Emissions (annual)
  - 72,000 tonnes CO<sub>2</sub>-eq.



# University of Calgary System



# Heating load profile

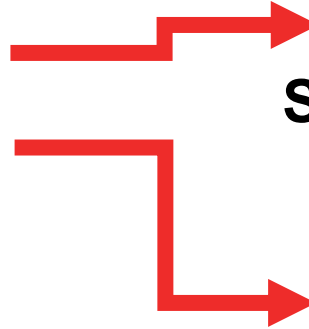


# Introduction

Cold water return



Natural Gas Turbine

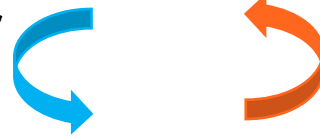


Supplemental Boilers



Electricity

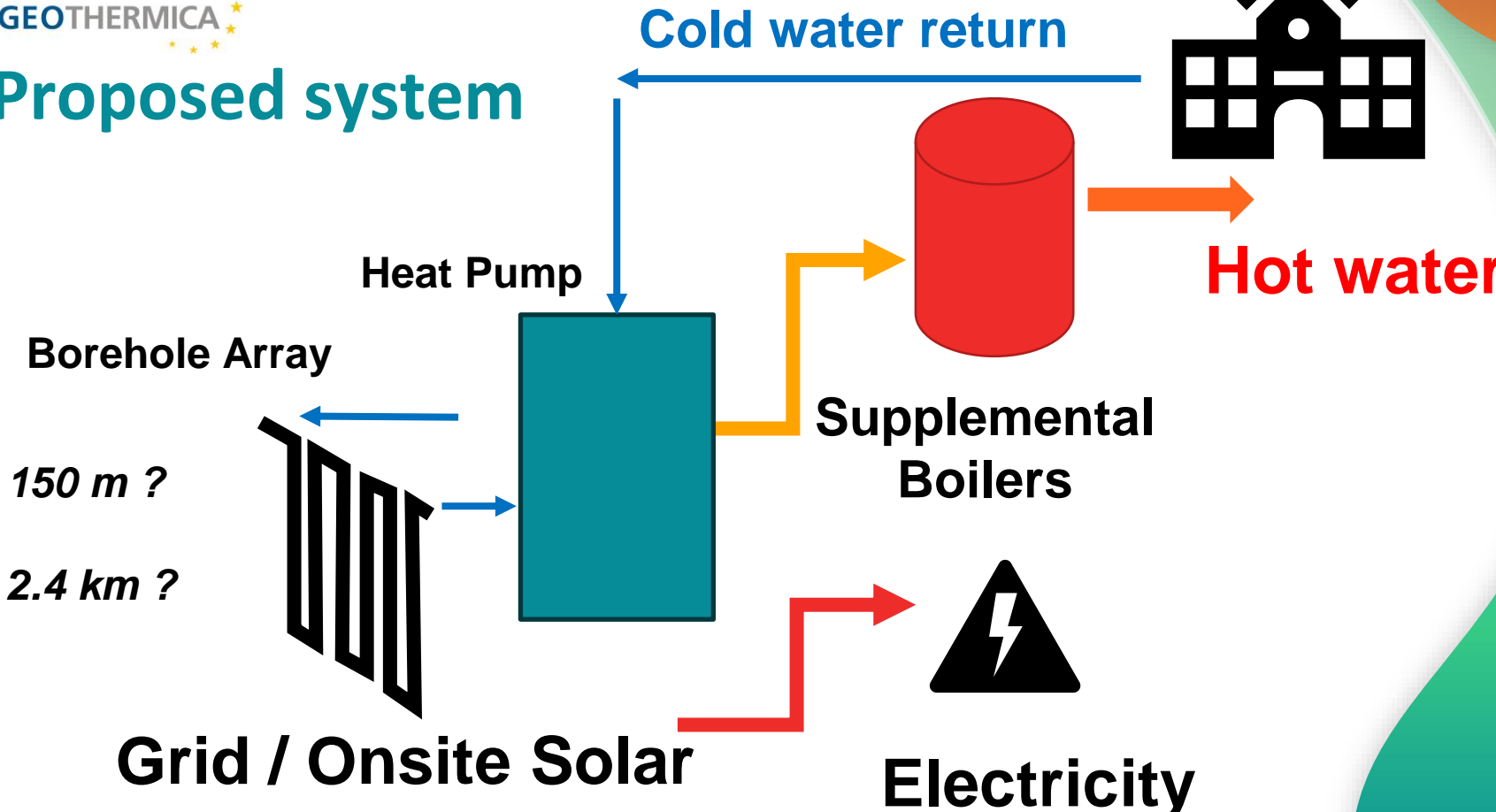
Heat Exchanger



Hot water

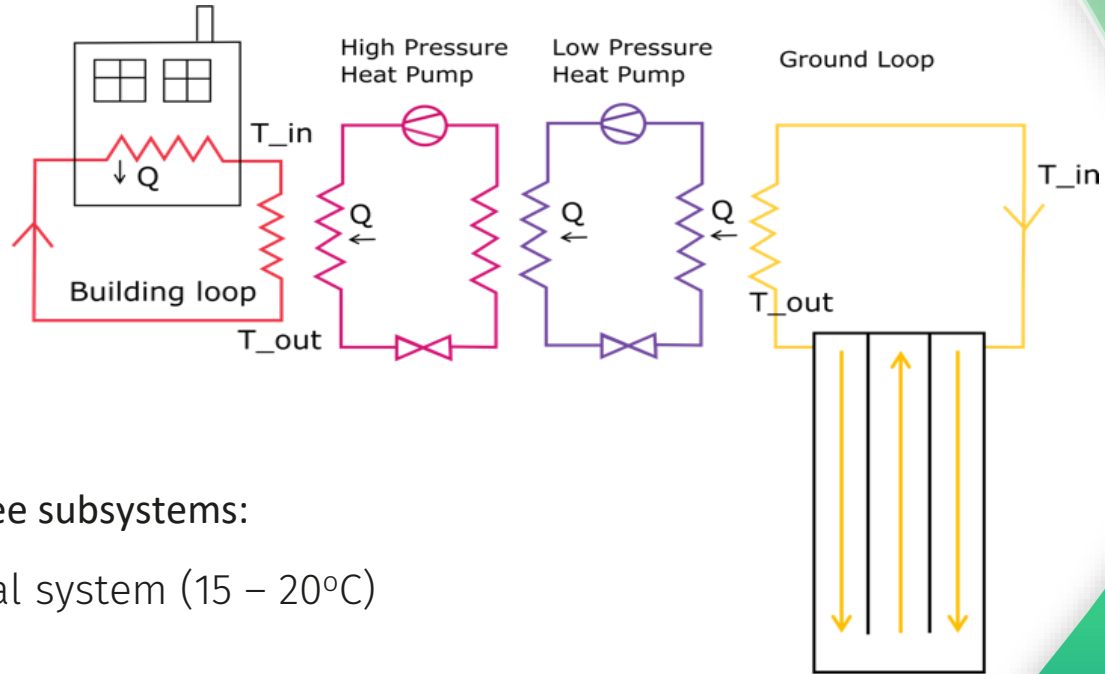


# Proposed system



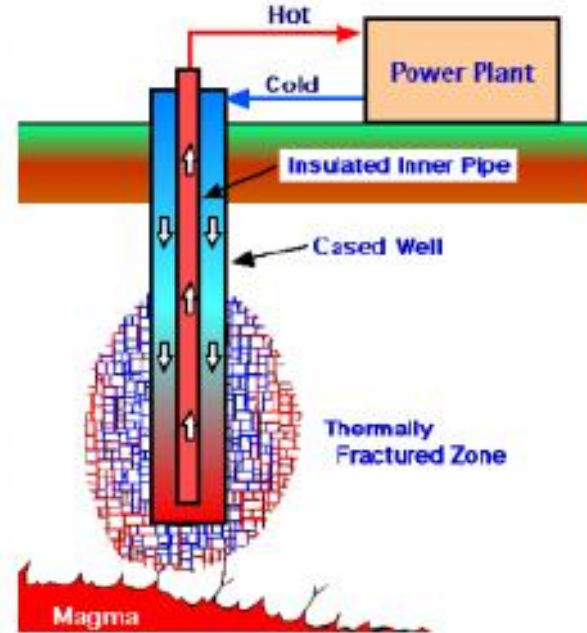


# System design

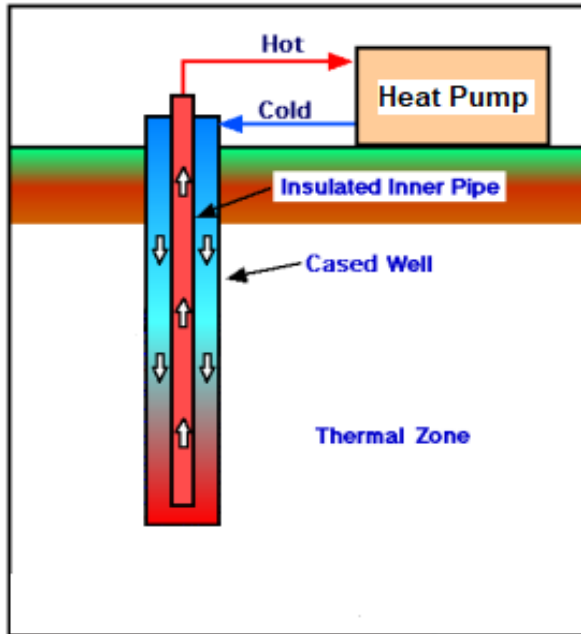


- Overall system integrates three subsystems:
  - Subsurface geothermal system (15 – 20°C)
  - Heat pump
  - Building heat loop (90-77°C)

- Co-axial closed loop
- Reservoir:
  - Permeable water-saturated sedimentary
  - $T_{res} = 60^{\circ}\text{C}$  @ 2000m
- Piping:
  - 11" ID Casing, 3" ID / 3.5" OD
- Operation:
  - $T_{in} = 15^{\circ}\text{C}$



Source: Morita et. al., 2005



Parameter	Value
Porosity (fraction)	0.01
$k_x = k_y = k_z$ (mD)	0.1
Thermal conductivity (geothermal reservoir) (J/mdayC)	274,000
Thermal conductivity (overburden) (J/mdayC)	210,000
Heat capacity (geothermal reservoir) (J/m <sup>3</sup> C)	$2.6 \times 10^6$
Heat capacity (overburden) (J/m <sup>3</sup> C)	$2 \times 10^6$
Tubing insulation thickness (in)	0.5
Tubing ID (in)	3
Tubing OD (in)	3.5
Casing ID (in)	11
Casing OD (in)	13.8
Hole diameter (in)	13.8 (no cement)
P at 2000 m (kPa)	20,000
T at 0 m (°C)	15
T at 2000 m (°C)	60

# Sub-surface simulation results

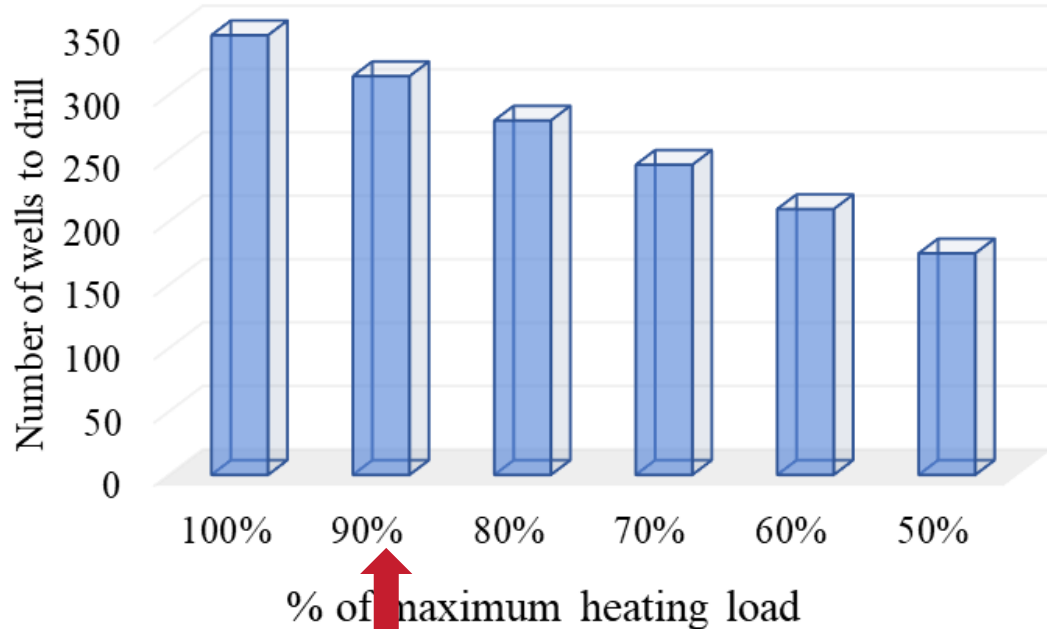
Injection Rate (m <sup>3</sup> /d)	Produced water temperature at surface (°C)			Thermal Output (kW)		
	1 year	5 years	10 years	1 year	5 years	10 years
100	30	28.6	28.1	73	66	63
500	19.4	18.9	18.7	106	94	90
1000	17.3	17	16.9	111	97	92
5000	15.4	15.4	15.4	97	97	97

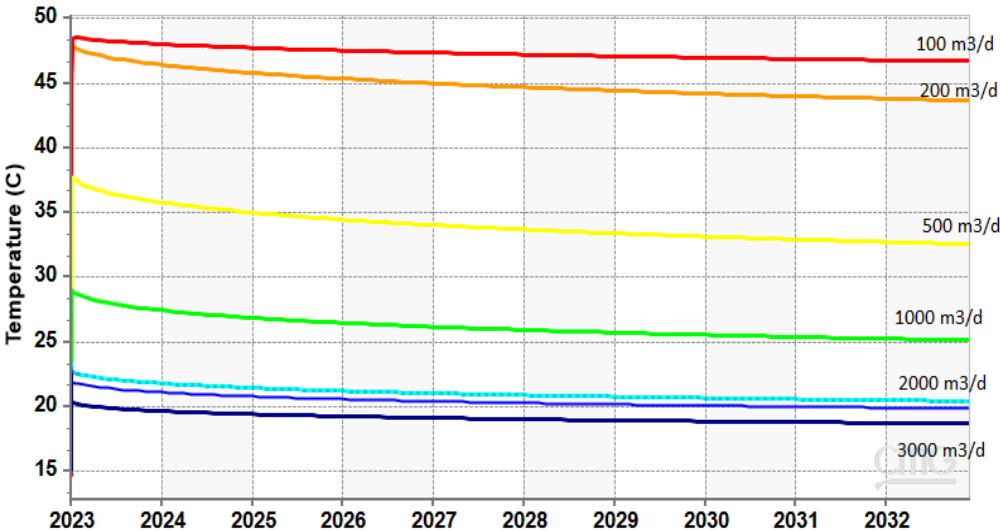
- Results (without the seasonal thermal energy storage) show an evident decline of working fluid's return temperature → indicating depletion of thermal reservoir with time!

# Sub-surface system design

- Optimal flow rate of 500 m<sup>3</sup>/d, results in the temperature gain of 4° C.
- To meet 90% of the peak heating load, about **300** vertical wellbores need to be drilled.

Number of vertical well bores required at 500 m<sup>3</sup>/d



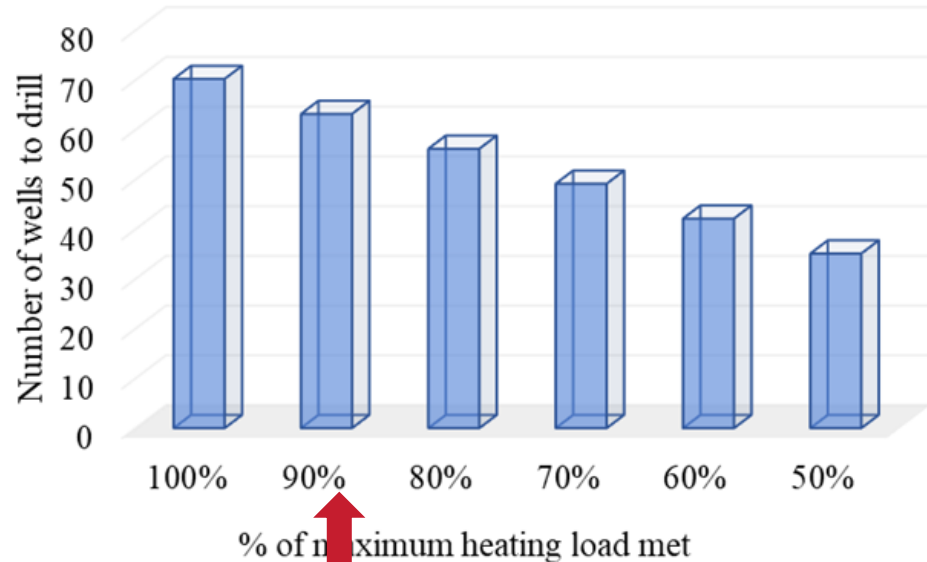


Flow rate (m <sup>3</sup> /day)	Produced temperature (°C)	Thermal output (kW)
100	46.7	153
200	43.7	278
500	32.6	426
1000	25.2	493
2000	20.4	523
2250	19.9	533
3000	18.7	537

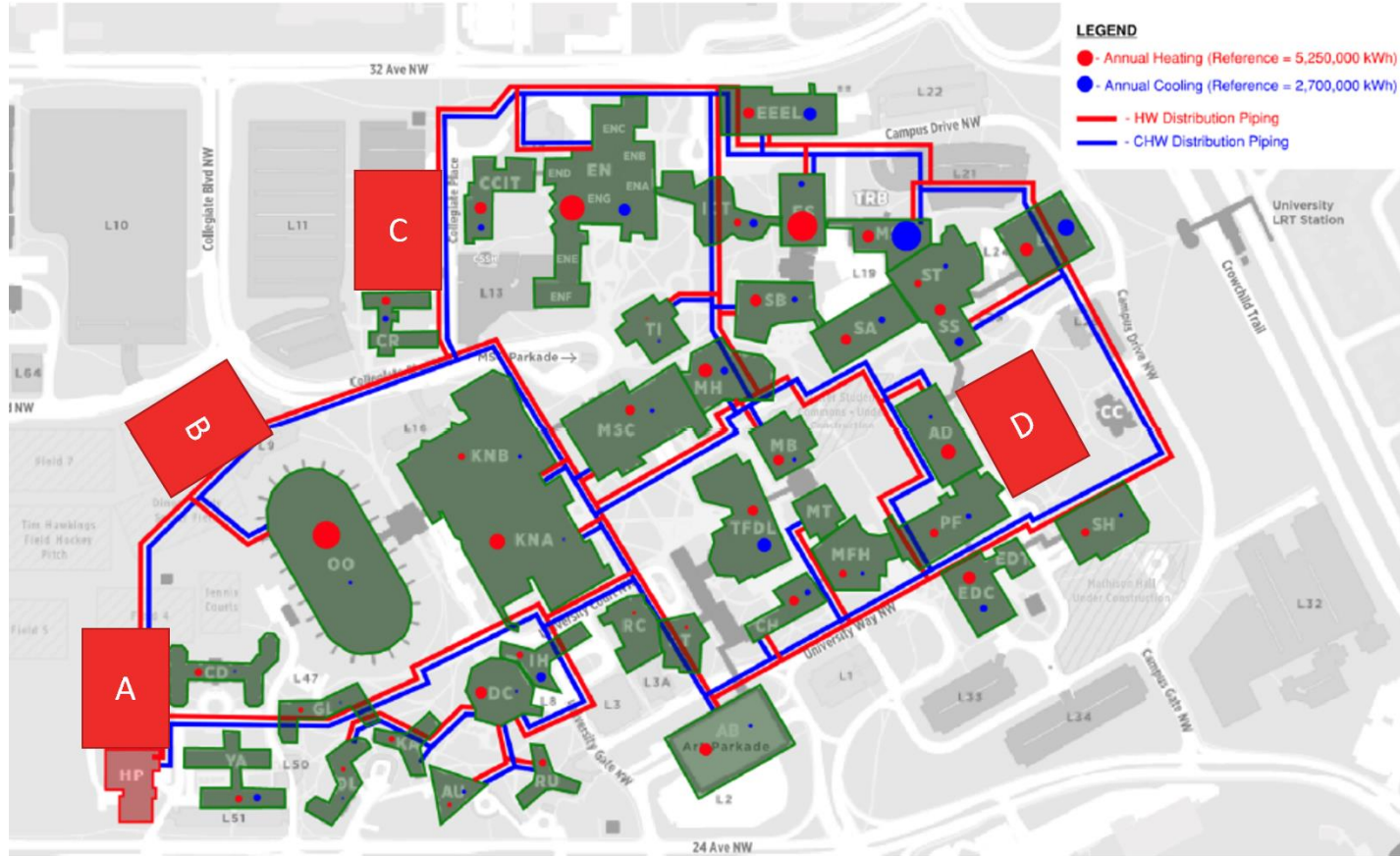
- Drilled to 2 km depth and then extended laterally for another 2 km.

- The case of 1000 m<sup>3</sup>/d is favourable, as it yields a  $\Delta T$  of 9°C.
- To meet 90% of the peak heating load, about **60** horizontal wellbores need to be drilled.

Number of horizontal well bores required at 1000 m<sup>3</sup>/d

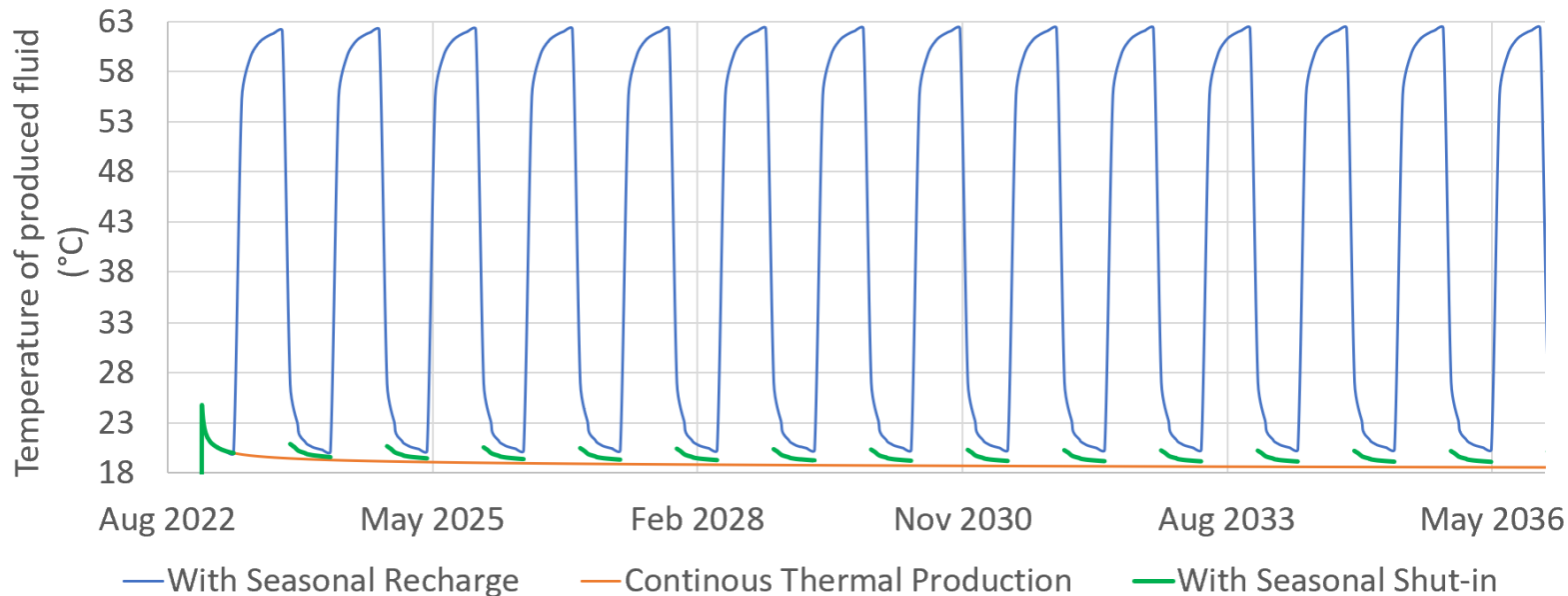


# Potential drilling sites



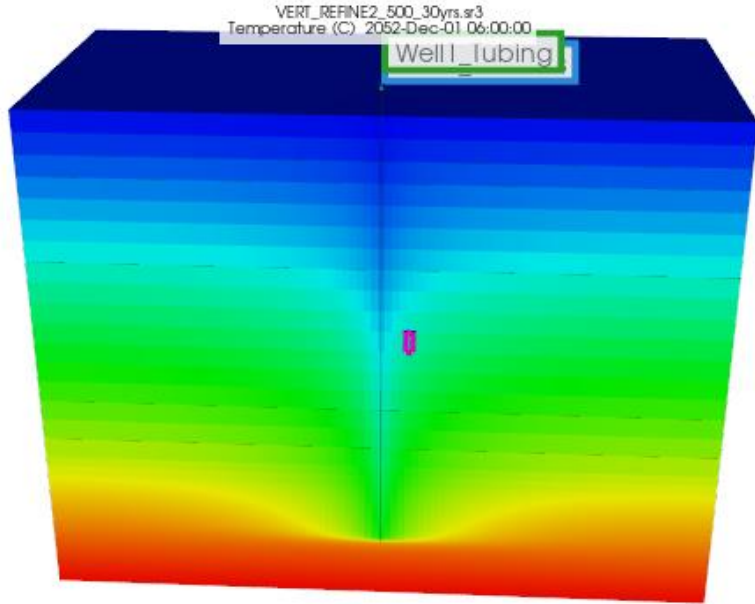


## Seasonal Recharge

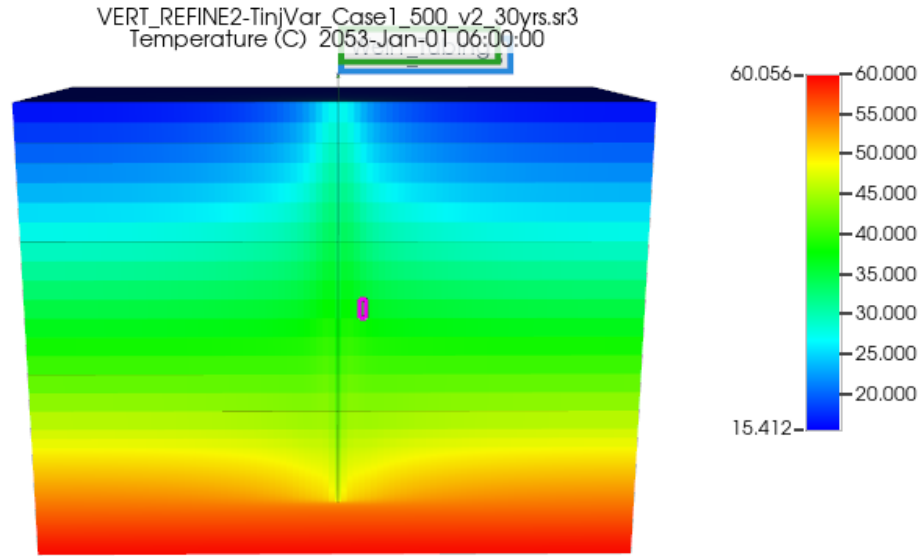


## Seasonal Recharge

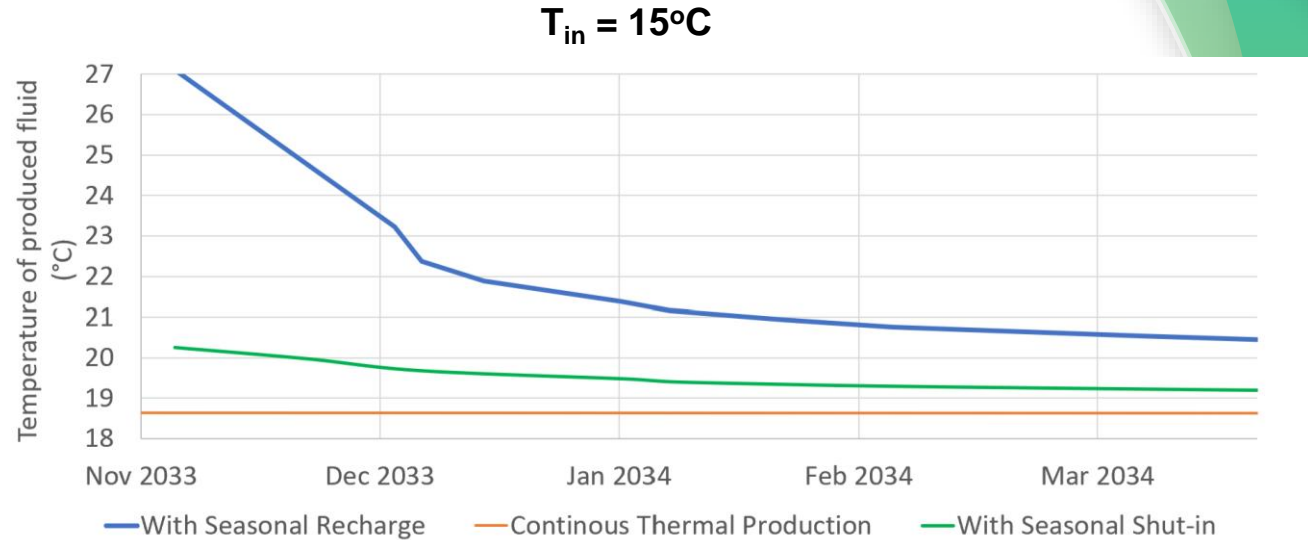
### Without Recharge



### With Recharge



## Seasonal Recharge



Winter Operation:

$T_{in} = 15^{\circ}\text{C}$

$Q = 500 \text{ m}^3/\text{day}$  (3145 BBL/day)

Summer Operation:

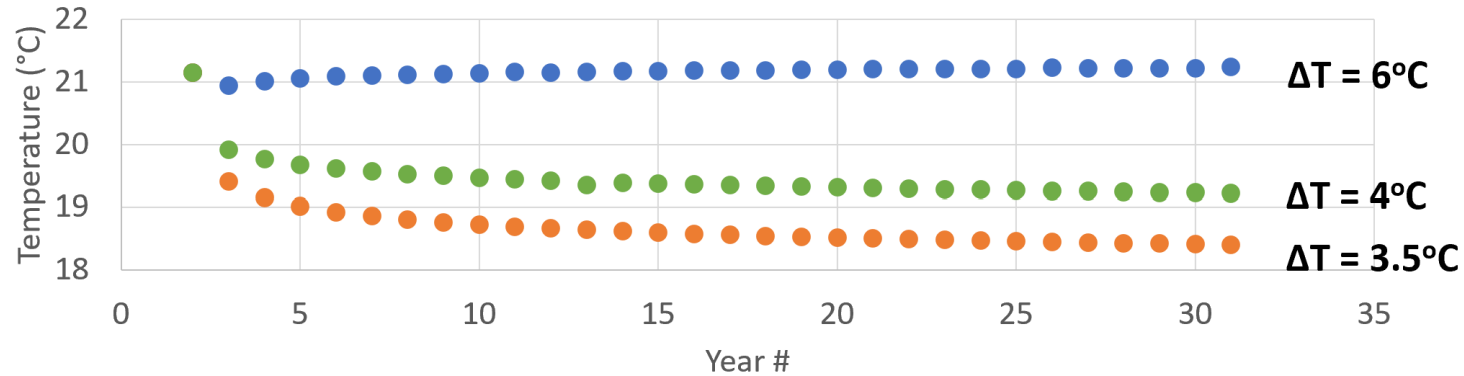
$T_{in} = 70^{\circ}\text{C}$

$Q = 500 \text{ m}^3/\text{day}$

## Seasonal Recharge

$T_{in} = 15^{\circ}C$

Temperature of Produced Fluids in Feb



● With Seasonal Recharge    ● Continuous Production    ● With Seasonal Shut-in

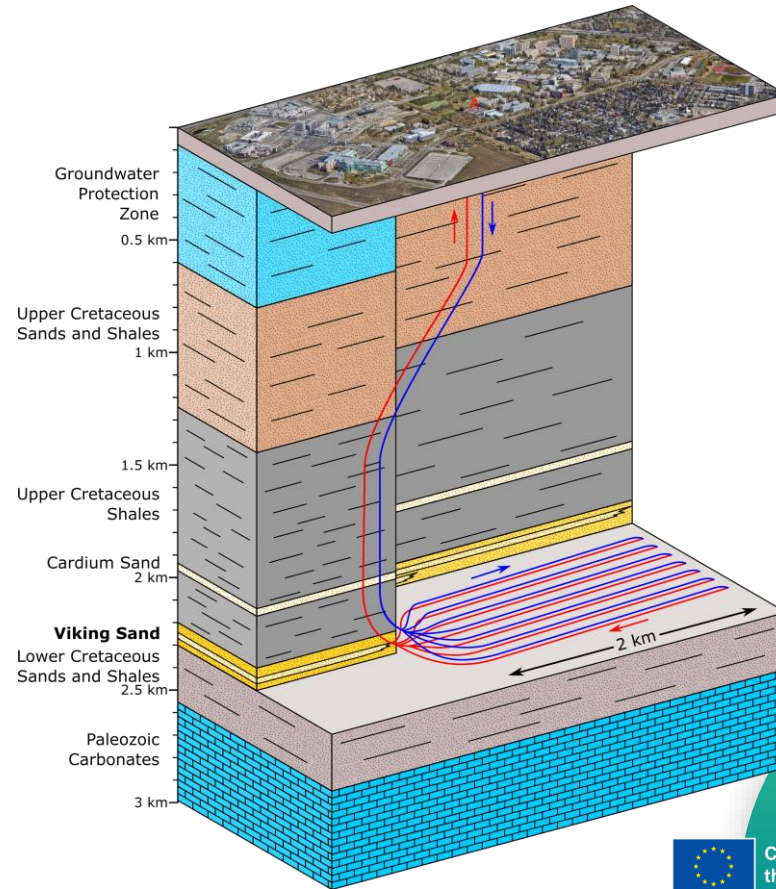
Small difference, but **50-70% increase in thermal power!**

## Seasonal Recharge

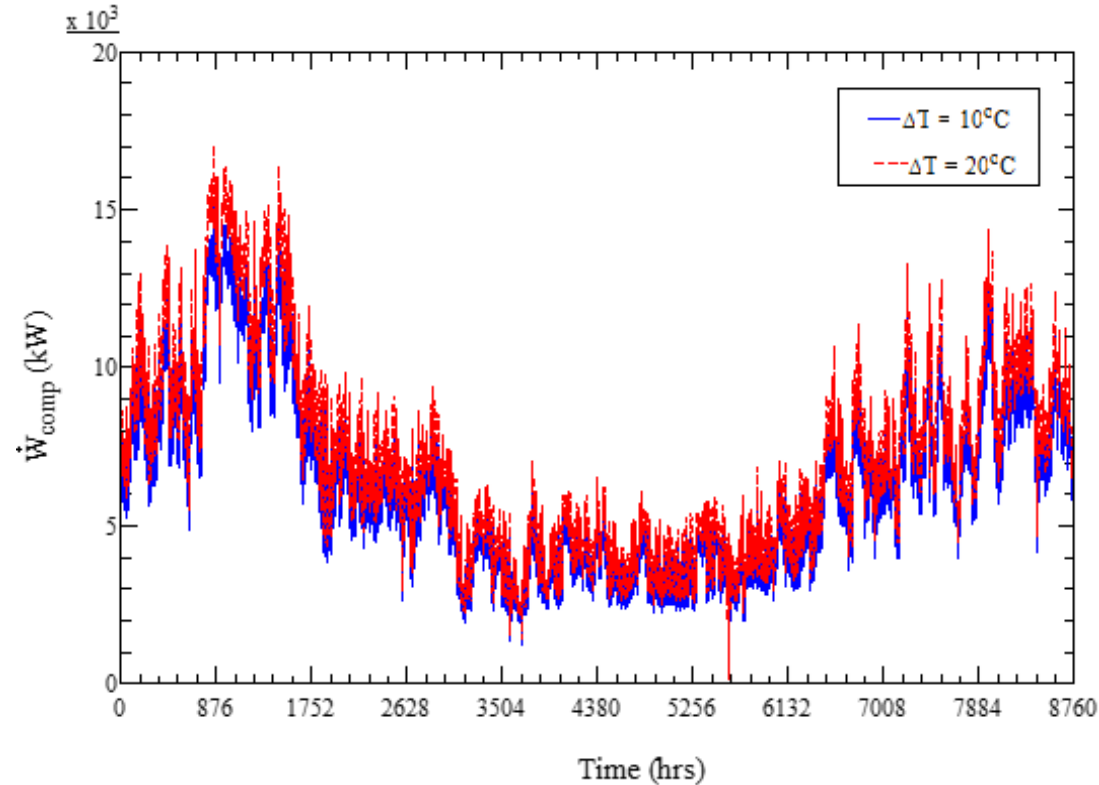
Potential to offset **90%+** of heating load with 30 horizontal wells pairs in a **closed-loop** design.

Why closed-loop?

***H<sub>2</sub>S ... an Alberta problem***



# Heat pump performance



- Refrigerant: R123zd(E)
- Supply Temperature:  $95^\circ\text{C}$
- Return Temperature:  $75^\circ\text{C}$
- COP: 3.068 average for the year.
- Power needed: 111 kW to 17 MW

# Acknowledgements

Results in this presentation come from a study presented at the 2023 Geothermal Rising Conference. The contribution of each author is duly acknowledged.

## **Reservoir thermal storage as part of a middle-deep closed-loop geothermal system**

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## Government Partners



## Federal and Provincial Funders



## Industry Partners





# Thank you

